## Important correlations figures and tables for MEP365 Thermal Measurements 2021

## Ch. 1 Introduction

### 1.1 Instrument uncertainty

$u_{c}=\sqrt{\left(e_{1}{ }^{2}+e_{2}{ }^{2}+e_{3}{ }^{2}+\ldots . e_{m}{ }^{2}\right)}$
Where $\mathrm{e}_{1}, \mathrm{e}_{2}, \ldots$. are the errors

## Ch. 2 \& Ch. 3 Signals \& Response of a Measurement System

## A) Signals

| Deterministic functions | Non-deterministic functions |
| :---: | :---: |
|  <br> Figure 2.5 Examples of dynamic signals. |  |

Signal average and RMS (Root Mean Squared)
Average

$$
\bar{y}=\frac{\int_{t_{1}}^{t_{2}} y(t) d t}{\int_{t_{1}}^{t_{2}} d t}
$$

RMS (Root Mean Squared) $\quad y_{r m s}=\sqrt{\frac{1}{t_{2}-t_{1}} \int_{t_{1}}^{t_{2}} y^{2} d t}$
Sinusoidal wave

$$
y(t)=A \cos (\omega t)+B \sin (\omega t)
$$

Period T [s] \& frequency $f[\mathrm{~Hz}]$

$$
T=\frac{2 \pi}{\omega}=\frac{1}{f}
$$

The combined sine and cosine function can be written in either sine or cosine wave:

$$
\begin{gathered}
y(t)=A \cos (\omega t)+B \sin (\omega t) \\
y(t)=C \cos (\omega t-\phi) \\
y(t)=C \sin \left(\omega t+\phi^{*}\right)
\end{gathered}
$$

where
$C=\sqrt{A^{2}+B^{2}}$ and $\quad \phi=\tan ^{-1}\left(\frac{B}{A}\right), \quad \phi^{*}=\tan ^{-1}\left(\frac{A}{B}\right), \quad \phi^{*}=\frac{\pi}{2}-\phi$
B) System response


General form of measuring system differential equation

$$
a_{n} \frac{d^{n} y}{d t^{n}}+a_{n-1} \frac{d^{n-1} y}{d t^{n-1}}+\cdots a_{1} \frac{d y}{d t}+a_{0} y=F(t)
$$

## B1- zero order system

$$
\begin{gathered}
a_{0} y=F(t) \\
y(t)=K F(t)
\end{gathered}
$$

$\mathrm{K}=1 / \mathrm{a}_{0}$ is called static sensitivity

## B2-First order system

$$
\begin{gathered}
a_{1} \frac{d y}{d t}+a_{0} y=F(t) \\
\frac{a_{1}}{a_{0}} \frac{d y}{d t}+y=\frac{1}{a_{0}} F(t) \\
\tau \frac{d y}{d t}+y=K F(t)
\end{gathered}
$$

$\tau$ is the time constant, which a fundamental characteristic of a first order system

## B2-a step response for first order system

Input step:

$$
F(t)=A U(t)
$$

$$
\tau \dot{y}+y=K F(t)=K A U(t)
$$

$\mathrm{U}(\mathrm{t})$ is the unit step
The solution is given by:

$$
y(t)=\underset{\text { Steady }}{K A} \quad+\quad \begin{gathered}
\left(y_{0}-K A\right) e^{-t / \tau} \\
\text { Transient part }
\end{gathered}
$$

The error fraction function is defined as

$$
\Gamma(t)=\frac{y(t)-y_{\infty}}{y_{0}-y_{\infty}}=e^{-t / \tau}
$$



B2-b Frequency response for the first order system

$$
\tau \dot{y}+y=K A \sin (\omega t)
$$

Transfer function

$$
G(s)=\frac{1}{1+\tau s}
$$

The general solution is given by:

$$
y(t)=C e^{-t / \tau}+\frac{K A}{\sqrt{1+(\omega t)^{2}}} \sin \left(\omega t-\tan ^{-1} \omega t\right)
$$

$$
\begin{aligned}
& y(t)=C e^{-t / \tau}+B(\omega) \sin [\omega t+\Phi] \\
& B(\omega)=\frac{K A}{\sqrt{1+(\omega \tau)^{2}}} \\
& \Phi(\omega)=-\tan ^{-1}(\omega \tau)
\end{aligned}
$$

Magnitude $\quad M(\omega)=\frac{B}{K A}=\frac{1}{\sqrt{1+(\omega \tau)^{2}}}$
Time delay $\boldsymbol{\beta}_{\mathbf{1}}, \beta_{1}=\frac{\phi}{\omega}$

| Magnitude | Phase shift |
| :---: | :---: |
|  |  |

## B3-Second order system

$$
\begin{gathered}
m \ddot{y}+c \dot{y}+k y=F(t) \\
a_{2} \dot{y}+a_{1} \dot{y}+a_{0} y=F(t) \\
\frac{1}{\omega_{n}^{2}} \ddot{y}+\frac{2 \zeta}{\omega_{n}} \dot{y}+y=K F(t)
\end{gathered}
$$

Natural frequency

$$
\begin{aligned}
& \omega_{n}=\sqrt{\frac{a_{0}}{a_{2}}}=\sqrt{\frac{k}{m}} \\
& \zeta=\frac{c}{c_{c}}=\frac{a_{1}}{2 \sqrt{a_{0} a_{2}}}=\frac{c}{2 \sqrt{k m}}
\end{aligned}
$$

Damping ratio
$B 3-a$ step response for a second order system
$y(t)=K A-K A e^{-\zeta \omega_{n} t}\left[\frac{\zeta}{\sqrt{1-\zeta^{2}}} \sin \left(\omega_{n} t \sqrt{1-\zeta^{2}}\right)+\cos \left(\omega_{n} t \sqrt{1-\zeta^{2}}\right)\right] \quad 0 \leq \zeta<1$
$y(t)=K A-K A\left(1+\omega_{n} t\right) e^{-\omega_{n} t}$
$\zeta=1$

Ringing frequency

$$
\begin{gathered}
T_{d}=\frac{2 \pi}{\omega_{d}}=\frac{1}{f_{d}} \\
\omega_{d}=\omega_{n} \sqrt{1-\zeta^{2}}
\end{gathered}
$$

| Step response | Rise time, settling time and Ringing frequency for under damped system |
| :---: | :---: |
|  <br> Figure 3.14 Second-order system time response to a step function input. |  |



## B3-b Frequency Response for the second order system due periodic input

Transfer function

$$
G(j \omega)=\frac{1}{\left[1-\left(\frac{\omega}{\omega_{n}}\right)^{2}+\left(\frac{2 \zeta \omega}{\omega_{n}}\right) j\right]}
$$

Magnitude:

$$
M(\omega)=\frac{B(\omega)}{K A}=\frac{1}{\left\{\left[1-\left(\frac{\omega}{\omega_{n}}\right)^{2}\right]^{2}+\left[2 \zeta \omega / \omega_{n}\right]^{2}\right\}^{1 / 2}}
$$

Phase shift:

$$
\phi(\omega)=\tan ^{-1}\left(-\frac{2 \zeta \omega / \omega_{n}}{1-\left(\frac{\omega}{\omega_{n}}\right)^{2}}\right)
$$

Resonance frequency $\omega_{R}=\omega_{n} \sqrt{1-2 \zeta^{2}}$

Dynamic error $\quad \delta(\omega)=M(\omega)-1$
Magnitude

## Ch. 4 Probability and Statistics

if $x^{\prime}$ is the true value, $\bar{x}$ is the mean value and $u_{\bar{x}}$ is the uncertainty then the true value for certain probability is given by
$x^{\prime}=\bar{x} \pm u_{\bar{x}} \quad(P \%)$
Number of intervals K to generate frequency distribution
$K=1.87(N-1)^{0.4}+1 \mathrm{~N}$ is the number of data points. For very large value of N , use $K=N^{\frac{1}{2}}$ provided at least one interval with occurrences $\geq 5$ (i.e. $\mathrm{n}_{\mathrm{j}} \geq 5$ ).

### 4.1 Infinite statistics

If the probability density function $\mathrm{p}(\mathrm{x})$ is known in the absence of the systematic errors ( $x^{\prime}=\bar{x}$ ), then the true mean value can be found using
$x^{\prime}=\int_{-\infty}^{+\infty} x p(x) d x$
The variance is given by
$\sigma^{2}=\int_{-\infty}^{+\infty}(x-\bar{x})^{2} p(x) d x$
and the standard of deviation is $\sigma$
Normal (Gauss normal distribution function)
$p(x)=\frac{1}{\sigma \sqrt{2 \pi}} \exp \left[-\frac{1}{2} \frac{\left(x-x^{\prime}\right)}{\sigma^{2}}\right]$
Define $\mathrm{z}_{1}$ as
$z_{1}=\frac{x_{1}-x^{\prime}}{\sigma}$ and $\beta=\frac{\mathrm{x}-\mathrm{x}^{\prime}}{\sigma}$
Probability for z to be between $-\mathrm{z}_{1}$ and $\mathrm{z}_{1}$
$P\left(-z_{1} \leq z \leq+z_{1}\right)=2\left[\frac{1}{\sqrt{2 \pi}} \int_{0}^{z_{1}} e^{-\beta^{2} / 2} d \beta\right]$
The term between the two brackets above is called half sided integral. It is tabulated in the following table (Table 4.3)

The probability that the $\mathrm{i}^{\text {th }}$ measured value will have a value in the range $\mathrm{x}^{\prime} \pm \mathrm{z}_{1} \sigma$ is $2 \mathrm{P}\left(\mathrm{z}_{1}\right)^{*} 100$ $=\mathrm{P} \%$

$$
x_{i}=x^{\prime} \pm z_{1} \sigma
$$

Table 4.3 Probability Values for Normal Error Function
One-Sided Integral Solutions for $p\left(z_{1}\right)=\frac{1}{(2 \pi)^{1 / 2}} \int_{0}^{z_{1}} e^{-\beta^{2} / 2} \mathrm{~d} \beta$

| $z_{1}=\frac{x_{1}-x^{\prime}}{\sigma}$ | 0.00 | 0.01 | 0.02 | 0.03 | 0.04 | 0.05 | 0.06 | 0.07 | 0.08 | 0.09 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.0 | 0.0000 | 0.0040 | 0.0080 | 0.0120 | 0.0160 | 0.0199 | 0.0239 | 0.0279 | 0.0319 | 0.0359 |
| 0.1 | 0.0398 | 0.0438 | 0.0478 | 0.0517 | 0.0557 | 0.0596 | 0.0636 | 0.0675 | 0.0714 | 0.0753 |
| 0.2 | 0.0793 | 0.0832 | 0.0871 | 0.0910 | 0.0948 | 0.0987 | 0.1026 | 0.1064 | 0.1103 | 0.1141 |
| 0.3 | 0.1179 | 0.1217 | 0.1255 | 0.1293 | 0.1331 | 0.1368 | 0.1406 | 0.1443 | 0.1480 | 0.1517 |
| 0.4 | 0.1554 | 0.1591 | 0.1628 | 0.1664 | 0.1700 | 0.1736 | 0.1772 | 0.1809 | 0.1844 | 0.1879 |
| 0.5 | 0.1915 | 0.1950 | 0.1985 | 0.2019 | 0.2054 | 0.2088 | 0.2123 | 0.2157 | 0.2190 | 0.222 |
| 0.6 | 0.2257 | 0.2291 | 0.2324 | 0.2357 | 0.2389 | 0.2422 | 0.2454 | 0.2486 | 0.2517 | 0.2549 |
| 0.7 | 0.2580 | 0.2611 | 0.2642 | 0.2673 | 0.2704 | 0.2734 | 0.2764 | 0.2794 | 0.2823 | 0.2852 |
| 0.8 | 0.2881 | 0.2910 | 0.2939 | 0.2967 | 0.2995 | 0.3023 | 03051 | 0.3078 | 0.3106 | 0.3133 |
| 0.9 | 0.3159 | 0.3186 | 0.3212 | 0.3238 | 0.3264 | 0.3289 | 03315 | 0.3340 | 0.3365 | 0.3389 |
| 1.0 | 0.3413 | 0.3438 | 0.3461 | 0.3485 | 0.3508 | 0.3531 | 03554 | 0.3577 | 0.3599 | 0.3621 |
| 1.1 | 0.3643 | 03665 | 0.3686 | 0.3708 | 0.3729 | 0.3749 | 03770 | 0.3790 | 0.3810 | 0.3830 |
| 1.2 | 0.3849 | 03869 | 0.3888 | 0.3907 | 0.3925 | 0.3944 | 03962 | 0.3980 | 0.3997 | 0.4015 |
| 1.3 | 0.4032 | 0.4049 | 0.4066 | 0.4082 | 0.4099 | 0.4115 | 0.4131 | 0.4147 | 0.4162 | 0.417 |
| 1.4 | 0.4192 | 0.4207 | 0.4222 | 0.4236 | 0.4251 | 0.4265 | 0.4279 | 0.4292 | 0.4306 | 0.4319 |
| 1.5 | 0.4332 | 0.4345 | 0.4357 | 0.4370 | 0.4382 | 0.4394 | 0.4406 | 0.4418 | 0.4429 | 0.4441 |
| 1.6 | 0.4452 | 0.4463 | 0.4474 | 0.4484 | 0.4495 | 0.4505 | 0.4515 | 0.4525 | 0.4535 | 0.4545 |
| 1.7 | 0.4554 | 0.4564 | 0.4573 | 0.4582 | 0.4591 | 0.4599 | 0.4608 | 0.4616 | 0.4625 | 0.4633 |
| 1.8 | 0.4641 | 0.4649 | 0.4656 | 0.4664 | 0.4671 | 0.4678 | 0.4686 | 0.4693 | 0.4699 | 0.4706 |
| 1.9 | 0.4713 | 0.4719 | 0.4726 | 0.4732 | 0.4738 | 0.4744 | 0.4750 | 0.4758 | 0.4761 | 0.4767 |
| 2.0 | 0.4772 | 0.4778 | 0.4783 | 0.4788 | 0.4793 | 0.4799 | 0.4803 | 0.4808 | 0.4812 | 0.4817 |
| 2.1 | 0.4821 | 0.4826 | 0.4830 | 0.4834 | 0.4838 | 0.4842 | 0.4846 | 0.4850 | 0.4854 | 0.4857 |
| 2.2 | 0.4861 | 0.4864 | 0.4868 | 0.4871 | 0.4875 | 0.4878 | 0.4881 | 0.4884 | 0.4887 | 0.4890 |
| 2.3 | 0.4893 | 0.4896 | 0.4898 | 0.4901 | 0.4904 | 0.4906 | 0.4909 | 0.4911 | 0.4913 | 0.4916 |
| 2.4 | 0.4918 | 0.4920 | 0.4922 | 0.4925 | 0.4927 | 0.4929 | 0.4931 | 0.4932 | 0.4934 | 0.4936 |
| 2.5 | 0.4938 | 0.4940 | 0.4941 | 0.4943 | 0.4945 | 0.4946 | 0.4948 | 0.4949 | 0.4951 | 0.4952 |
| 2.6 | 0.4953 | 0.4955 | 0.4956 | 0.4957 | 0.4959 | 0.4960 | 0.4961 | 0.4962 | 0.4963 | 0.4964 |
| 2.7 | 0.4965 | 0.4966 | 0.4967 | 0.4968 | 0.4969 | 0.4970 | 0.4971 | 0.4972 | 0.4973 | 0.4974 |
| 2.8 | 0.4974 | 0.4975 | 0.4976 | 0.4977 | 0.4977 | 0.4978 | 0.4979 | 0.4979 | 0.4980 | 0.4981 |
| 2.9 | 0.4981 | 0.4982 | 0.4982 | 0.4983 | 0.4984 | 0.4984 | 0.4985 | 0.4985 | 0.4986 | 0.4986 |
| 3.0 | 0.49865 | 0.4987 | 0.4987 | 0.4988 | 0.4988 | 0.4988 | 0.4989 | 0.4989 | 0.4989 | 0.4990 |




### 4.2 Finite statistics

Sample mean, $\bar{x}$
$\bar{x}=\frac{1}{N} \sum_{i=1}^{N} x_{i}$
Sample variance, $s_{x}{ }^{2}$
$s_{x}{ }^{2}=\frac{1}{N-1} \sum_{i=1}^{N}\left(x_{i}-\bar{x}\right)^{2}$
Sample standard of deviation, $s_{x}$
$s_{x}=\sqrt{s_{x}{ }^{2}}$
Standard deviation of the mean
$s_{\bar{x}}=\frac{s_{x}}{\sqrt{N}}$
Sample data interval for certain probability $\quad \bar{x} \pm t_{v, P} s_{x} \quad(P \%)$

True mean value estimation with probability $\mathrm{P}(\%) \quad \bar{x} \pm t_{v, P} s_{\bar{x}} \quad(P \%)$
$t_{v, P}$ is the $t$-estimator which can be found from table 4.4 below.
v is the degree of freedom $=\mathrm{N}-1$

Table 4.4 Student's $t$ Distribution

| $v$ | $t_{50}$ | $t_{90}$ | $t_{95}$ | $t_{99}$ |
| ---: | :---: | :---: | :---: | :---: |
| 1 | 1.000 | 6.314 | 12.706 | 63.657 |
| 2 | 0.816 | 2.920 | 4.303 | 9.925 |
| 3 | 0.765 | 2.353 | 3.182 | 5.841 |
| 4 | 0.741 | 2.132 | 2.770 | 4.604 |
| 5 | 0.727 | 2.015 | 2.571 | 4.032 |
| 6 | 0.718 | 1.943 | 2.447 | 3.707 |
| 7 | 0.711 | 1.895 | 2.365 | 3.499 |
| 8 | 0.706 | 1.860 | 2.306 | 3.355 |
| 9 | 0.703 | 1.833 | 2.262 | 3.250 |
| 10 | 0.700 | 1.812 | 2.228 | 3.169 |
| 11 | 0.697 | 1.796 | 2.201 | 3.106 |
| 12 | 0.695 | 1.782 | 2.179 | 3.055 |
| 13 | 0.694 | 1.771 | 2.160 | 3.012 |
| 14 | 0.692 | 1.761 | 2.145 | 2.977 |
| 15 | 0.691 | 1.753 | 2.131 | 2.947 |
| 16 | 0.690 | 1.746 | 2.120 | 2.921 |
| 17 | 0.689 | 1.740 | 2.110 | 2.898 |
| 18 | 0.688 | 1.734 | 2.101 | 2.878 |
| 19 | 0.688 | 1.729 | 2.093 | 2.861 |
| 20 | 0.687 | 1.725 | 2.086 | 2.845 |
| 21 | 0.686 | 1.721 | 2.080 | 2.831 |
| 30 | 0.683 | 1.697 | 2.042 | 2.750 |
| 40 | 0.681 | 1.684 | 2.021 | 2.704 |
| 50 | 0.680 | 1.679 | 2.010 | 2.679 |
| 60 | 0.679 | 1.671 | 2.000 | 2.660 |
| $\infty$ | 0.674 | 1.645 | 1.960 | 2.576 |
|  |  |  |  |  |

## Chauvenet's criterion for outlier data

Let $z_{0}$ be $z_{0}=\left|\frac{x_{i}-\bar{x}}{s_{x}}\right|$
If $\left(1-2 * P\left(z_{0}\right)\right)<\frac{1}{2 N}$ then it can be considered outlier
Number of measurements required
$N_{T} \approx\left(\frac{t_{N_{1}-1,95} s_{1}}{d}\right)^{2} \quad \mathrm{P}=95 \%$
Additional data needed $\mathrm{N}_{\mathrm{T}}-\mathrm{N}_{1}$
$\mathrm{d}=\mathrm{CI} / 2$ where
CI is the confidence interval


## Least squares method

A polynomial of order m between y and x is given by:

$$
\begin{aligned}
& y_{c}=a_{0}+a_{1} x+a_{2} x^{2}+ \\
& +a_{m} x^{m} \\
& D=\sum_{i=1}^{N}\left[y_{i}-\left(a_{0}+a_{1} x+a_{2} x^{2}+\ldots a_{m} x^{m}\right)\right]^{2} \\
& \frac{\partial D}{\partial a_{0}}=0=\frac{\partial}{\partial a_{0}}\left\{\sum_{i=1}^{N}\left[y_{i}-\left(a_{0}+a_{1} x+a_{2} x^{2}+\ldots a_{m} x^{m}\right)\right]^{2}\right\} \\
& \frac{\partial D}{\partial a_{1}}=0=\frac{\partial}{\partial a_{1}}\left\{\sum_{i=1}^{N}\left[y_{i}-\left(a_{0}+a_{1} x+a_{2} x^{2}+\ldots a_{m} x^{m}\right)\right]^{2}\right\} \\
& \frac{\partial D}{\partial a_{2}}=0=\frac{\partial}{\partial a_{2}}\left\{\sum_{i=1}^{N}\left[y_{i}-\left(a_{0}+a_{1} x+a_{2} x^{2}+\ldots a_{m} x^{m}\right)\right]^{2}\right\} \\
& \cdots \cdots .\left[\begin{array}{ccc}
N & \sum_{i=1}^{N} x_{i} & \sum_{i=1}^{N} x_{i}{ }^{2} \\
\sum_{i=1}^{N} x_{i} & \sum_{i=1}^{N} x_{i}{ }^{2} & \sum_{i=1}^{N} x_{i}{ }^{3} \\
\sum_{i=1}^{N} x_{i}{ }^{2} & \sum_{i=1}^{N} x_{i}{ }^{3} & \sum_{i=1}^{N} x_{i}^{4}
\end{array}\right]\left[\begin{array}{c}
a_{0} \\
a_{1} \\
a_{2}
\end{array}\right]=\left[\begin{array}{c}
\sum_{i=1}^{N} y_{i} \\
\sum_{i=1}^{N} x_{i} y_{i} \\
\sum_{i=1}^{N} x_{i}{ }^{2} y_{i}
\end{array}\right]
\end{aligned}
$$

Standard error of the fit: $\quad s_{y x}=\sqrt{\frac{\sum_{i}^{N}\left(y_{i}-y_{c i}\right)^{2}}{v}}$

Correlation coefficient,
$r=\sqrt{1-\frac{s_{y x}^{2}}{s_{y}^{2}}}$
Coefficient of determination $r^{2}$

Degree of freedom: $v=N-(m+1)$
Uncertainty of the fit $u= \pm t_{v, P} \frac{s_{y x}}{\sqrt{N}}$

## Ch. 5 Uncertainty

$$
x^{\prime}=\bar{x} \pm u_{x} \quad(P \%)
$$

## Design stage uncertainty

$u_{d}=\sqrt{u_{o}^{2}+u_{c}^{2}}$
$\mathrm{u}_{\mathrm{o}}=$ interpolation error=(1/2) resolution
$\mathrm{u}_{\mathrm{c}}=$ instrumental error


## Error Propagation

$$
\begin{aligned}
& \mathrm{R}=\mathrm{R}\left(\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}, \ldots \ldots \mathrm{x}_{\mathrm{L}}\right)_{]^{1 / 2}} \\
& u_{R}= \pm\left[\sum_{i=1}^{L}\left(\theta_{i} u_{x i}\right)^{2}\right]^{1 / 2} \quad \theta_{i}=\left(\frac{\partial R}{\partial x_{i}}\right)
\end{aligned}
$$

## Error Propagation using Numerical Approach

$$
\begin{aligned}
& R=R\left(x_{1}, x_{2}, x_{3}, \ldots . x_{L}\right) \quad R_{o}=R\left(\bar{x}_{1}, \bar{x}_{2}, \bar{x}_{3}, \ldots . \bar{x}_{L}\right) \\
& R_{i}^{+}=R\left(x_{i}+u_{x 1}, x_{2}, x_{3}, \ldots \ldots x_{L}\right) \quad R_{i}^{-}=R\left(x_{i}-u_{x 1}, x_{2}, x_{3}, \ldots \ldots x_{L}\right) \\
& \delta R_{i}^{+}=R_{i}^{+}-R_{o} \quad \delta R_{i}^{-}=R_{i}^{-}-R_{o} \\
& \quad \delta R_{i}=\frac{\delta R_{i}^{+}-\delta R_{i}^{-}}{2}=\theta_{i} u_{i} \quad u_{R}= \pm\left[\sum_{i=1}^{L}\left(\delta R_{i}\right)^{2}\right]^{1 / 2}
\end{aligned}
$$

Procedure to find the uncertainty for multiple measurements based on grouping the elemental errors into Bias uncertainty $b$, and random uncertainty s

1-Perfom multiple measurements for x
2-Identfy elemental errors $\mathrm{e}_{\mathrm{k}}$
3-For each $\mathrm{e}_{\mathrm{k}}$ assign $\left(b_{\bar{x}}\right)_{k}$ and $\left(s_{\bar{x}}\right)_{k}$
4-For each measurement, the standard random uncertainty is given by


Figure 5.6 Multiple-measurement uncertainty procedure for combining uncertainties.
$s_{\bar{x}}=\frac{s_{x}}{\sqrt{N}}$
5-Combining the systematic and the random uncertainties into

$$
\begin{aligned}
& b_{\bar{x}}=\left[\left(b_{\bar{x}}^{2}\right)_{1}+\left(b_{\bar{x}}^{2}\right)_{2}+\left(b_{\bar{x}}^{2}\right)_{3}+\cdots\left(b_{\bar{x}}^{2}\right)_{k}\right]^{\frac{1}{2}} \\
& s_{\bar{x}}=\left[\left(s_{\bar{x}}^{2}\right)_{1}+\left(s_{\bar{x}}^{2}\right)_{2}+\left(s_{\bar{x}}^{2}\right)_{3}+\cdots \cdot\left(s_{\bar{x}}^{2}\right)_{k}\right]^{\frac{1}{2}}
\end{aligned}
$$

6-The expanded uncertainty is evaluated using

$$
u_{x}=t_{v, P}\left[\left(b_{\bar{x}}\right)^{2}+\left(s_{\bar{x}}\right)^{2}\right]^{\frac{1}{2}}
$$

where the degree of freedom is found using

$$
v=\frac{\left(\sum_{k=1}^{K}\left(s_{\bar{x}}^{2}\right)_{k}+\left(b_{\bar{x}}^{2}\right)_{k}\right)^{2}}{\left.\left.\sum_{k=1}^{K}\left(s_{\bar{x}}^{4}\right)_{k} / v_{k}\right)+\sum_{k=1}^{K}\left(b_{\bar{x}}^{4}\right)_{k} / v_{k}\right)}
$$

The systematic part can be neglected in the above equation if it is very small.
Propagation of uncertainty to the results using the concept of grouping the errors into systematic and random errors

$$
\begin{aligned}
& \begin{array}{l}
R^{\prime}=\bar{R} \pm u_{R} \quad(\mathrm{P} \%) \\
u_{R}=f_{2}\left(b_{\bar{x} 1}, b_{\bar{x} 2},+b_{\bar{x} 3}, . . b_{\bar{x} L} ; s_{\bar{x} 1}, s_{\bar{x} 2}, \ldots . s_{\bar{x} L}\right)
\end{array} \\
& s_{R}=\left(\sum_{i=1}^{L}\left[\theta_{i} s_{\bar{x} i}\right]^{2}\right)^{1 / 2} \quad b_{R}=\left(\sum_{i=1}^{L}\left[\theta_{i} b_{\bar{x} i}\right]^{2}\right)^{1 / 2} \quad \theta_{i}=\left.\frac{\partial R}{\partial x_{i}}\right|_{x=\bar{x}} \\
& v_{R}=\frac{\left\{\sum_{i=1}^{L}\left(\theta_{i} s_{\bar{x} i}\right)^{2}\right\}^{2}}{\sum_{i-1}^{L}\left\{\left(\theta_{i} s_{\bar{x} i}\right)^{4} / v_{\bar{x} i}\right\}} \quad u_{R}=t_{v, P}\left[b_{R}^{2}+s_{R}^{2}\right]^{\frac{1}{2}}
\end{aligned}
$$

## Ch. 8 Temperature measurements

RTD
$R=R_{o}\left[1+\alpha\left(T-T_{o}\right)\right]$
For Platinum, $\alpha$ is $0.003927 \mathrm{C}^{-1}$

## Thermistors

$R=R_{o} e^{\beta\left[1 / T-1 / T_{o}\right]}$
Typical values of $\beta$ are between 3500 K and 4600 K .

## Thermocouple

- Seebeck effect
- Peltier effect
- Thomson effect

Tables for the variation of emf from standard thermocouple with $0^{\circ} \mathrm{C}$ reference junction are given at the end of these sheets. Temperatures are in ${ }^{\circ} \mathrm{C}$ and emf in mV .

## Conduction errors

$\frac{\theta(x)}{\theta_{w}}=\frac{\cosh (m x)}{\cosh (m L)}$ where $m^{2}=\frac{h P}{k A}$
$\frac{T_{p}-T_{\infty}}{T_{w}-T_{\infty}}=\frac{1}{\operatorname{Cosh}(m L)}$
Conduction error, $e_{c}=T_{p}-T_{\infty}=\frac{T_{w}-T_{\infty}}{\cosh (m L)}$

## Radiation errors

At equilibrium: heat by convection=heat by radiation
$q_{c}=q_{r}$

$$
h A_{p}\left(T_{\infty}-T_{p}\right)=F A_{p} \varepsilon_{p} \sigma\left[T_{p}^{4}-T_{w}^{4}\right]
$$

$\sigma=5.669 * 10^{-8} \mathrm{~W} /\left(\mathrm{m}^{2} . \mathrm{K}^{4}\right)$
Radiation error $\quad e_{r}=T_{p}-T_{\infty}$
Newton-Raphson's method for solving non-linear equations


$$
\begin{gathered}
T_{p, i+1}=T_{p, i}-\frac{f}{f^{\prime}} \\
\text { rror: }
\end{gathered}
$$

$e_{r}=T_{p}-T_{\infty}=\frac{F \varepsilon \sigma}{h}\left(T_{w}^{4}-T_{p}^{4}\right)$

## Recovery error (High speed flows)

Sound of speed in air is given by

$$
a=\sqrt{k R T}
$$

where
k is the specific heat ratio $\mathrm{C}_{\mathrm{p}} / \mathrm{C}_{\mathrm{v}}$
R ideal gas constant $=287 \mathrm{~J} / \mathrm{kg} . \mathrm{K}$
T is the temperature of air in Kelvin


Recovery error $e_{U}=T_{p}-T_{\infty}=\frac{r U^{2}}{2 C_{p}}$
for wires normal to flow $\quad r=0.68+0.07$
for wires parallel to flow $r=0.86+0.09$
Relation between probe temperature $T_{p}$ and stagnation temperature $T_{t}$
$T_{p}=T_{t}-\frac{(1-r) U^{2}}{2 C_{p}}$
$e_{U}=T_{t}-\frac{(1-r) U^{2}}{2 C_{p}}$
U is the flow speed
$T_{t}$ is stagnation temperature (total temperature), which can be found using
$\frac{U^{2}}{2}=C_{p}\left(T_{t}-T_{\infty}\right)$
$\mathrm{C}_{\mathrm{p}}$ must be in J/kg.K.

## Transient behavior of a temperature sensor

Time constant $\tau$ is given by $\quad \tau=\frac{\rho \forall C_{p}}{h A}$ where
$\forall$ is the volume of the sensor (or probe)
$\rho$ is the sensor (or probe) density
$\mathrm{C}_{\mathrm{p}}$ sensor specific heat
$h$ is the heat transfer between the sensor and the surrounding environment
A is the surface area of the sensor
For a probe initially at $\mathrm{T}=\mathrm{T}_{\mathrm{i}}$, subjected to environment at $\mathrm{T}_{\infty}$
$\tau \frac{d T}{d t}=\left(T_{\infty}-T\right)$
or using $\theta=\left(\mathrm{T}-\mathrm{T}_{\mathrm{i}}\right)$, and $\theta_{\infty}=\left(\mathrm{T}_{\infty}-\mathrm{T}_{\mathrm{i}}\right)$
$\frac{d \theta}{d t}+\frac{\theta}{\tau}=\frac{\theta}{\tau}$
and the solution
$\theta=\theta_{\infty}\left(1-e^{-t / \tau}\right)$

## Ch. 9 Pressure Measurements

$\gamma=$ Specific weight $=\rho \mathrm{g}\left[\mathrm{N} / \mathrm{m}^{3}\right]$
$\mathrm{S}=$ Specific gravity $=\rho / \rho_{\mathrm{w}}$ [Dimensionless].

## Straight U tube manometer

$\Delta p=\left(\rho_{m}-\rho\right) g H=\left(\gamma_{m}-\gamma\right) H$
$\rho_{\mathrm{m}}$ is the manometer fluid density, and $\rho$ is the fluid density

## Inclined manometer

$\Delta p=\left(\rho_{m}-\rho\right) g L \sin (\theta)=\left(\gamma_{m}-\gamma\right) L \sin (\theta)$
$\theta$ is the inclined angle of the manometer with the horizontal

## Deadweight tester



Gravity error for elevation z (in meter), and latitude angle $\phi$ (in degrees)
$e_{1}=-\left(2.637 * 10^{-3} \cos (2 \phi)+2.9 * 10^{-5} z+5 * 10^{-5}\right)$
Buoyancy effect
$e_{2}=-\gamma_{\text {air }} / \gamma_{\text {masses }}$
The indicated pressure is corrected using
$p=p_{i}\left(1+e_{1}+e_{2}\right)$

## Pitot static tube

$p_{v}=p_{t}-p_{x}=\frac{1}{2} \rho U_{x}^{2}$ or $U_{x}=\sqrt{\frac{2 \Delta p}{\rho}}$
For high speed gas
$U=\sqrt{2[k /(k-1)]\left[(p / \rho)^{(k-1) / k}-1\right]}$
k is specific heat ratio $\left(\mathrm{C}_{\mathrm{p}} / \mathrm{C}_{\mathrm{v}}\right)$

## Thermal Anemometry

$E^{2}=C+D U^{n}$
or
$E_{1}=K U$

## Doppler Anemometry

$f_{s}=f_{i}+f_{D}$
$U=\frac{\lambda}{2 \sin (\theta / 2)} f_{D}$

## Loading error

$\frac{E_{o}}{E_{i}}=\frac{1}{1+\left(R_{2} / R_{1}\right)\left[R_{1} / R_{m}+1\right]}$
When $\mathrm{R}_{\mathrm{m}} \rightarrow \infty$
$\frac{E_{o}}{E_{i}}=\frac{R_{1}}{R_{1}+R_{2}}$

$$
f_{o}=\frac{c \pm V_{o}}{c \mp V_{s}} f_{s}
$$

Doppler effect
$f_{o}$ observer frequency, $f_{s}$ source frequency, $V_{o}$ observer speed, $V_{s}$ source speed, $\mathrm{c}=$ speed of sound or light


## Ch. 10 Flow measurements

Flow rate through velocity determination
$Q=\iint_{A} U d A$
for circular pipe $\quad Q=2 \pi \sum U_{i j} r \Delta r$
Obstruction meters (orifice, venturi, and nozzle)
$Q_{I}=C E A_{o} \sqrt{\frac{2 \Delta p}{\rho}}=K_{o} A_{o} \sqrt{\frac{2 \Delta p}{\rho}}$
where
$\mathrm{Q}_{\mathrm{I}}$ is the volume flow rate assuming the flow to be incompressible
Velocity approach factor $E=\frac{1}{\sqrt{1-\beta^{4}}}, \beta=d_{o} / d_{1}$


Figure 10.4 Control volume concept as applied between two streamlines for flow through an obstruction meter.


C is the discharge coefficient= $\mathrm{f}\left(\operatorname{Re}_{\mathrm{d} 1}, \beta\right), \quad \operatorname{Re}_{d_{1}}=\frac{\rho \bar{U} d_{1}}{\mu}=\frac{\bar{U} d_{1}}{v}=\frac{4 Q}{\pi d_{1} v}$
$\mathrm{K}_{\mathrm{o}}=\mathrm{CE}=$ flow coefficient $=\mathrm{f}\left(\operatorname{Re}_{\mathrm{d} 1}, \beta\right)$, see Fig. 10.6 \& Fig. 10.11
the pressure drop using a manometer $\Delta p=\left(\gamma_{m}-\gamma\right) H$
$A_{o}=\frac{\pi d_{o}^{2}}{4}$
Compressibility effect
$Q=Q_{I} Y=C E A_{0} Y \sqrt{\frac{2 \Delta p}{\rho_{1}}}$
$\mathrm{Y}=$ expansion factor $=\mathrm{f}\left(\operatorname{Re}_{\mathrm{d} 1}, \beta\right)$, see Fig. 10.7
Compressibility effect is considered when $\left(p_{1}-p_{2}\right) / p_{1} \geq 0.1$
For Venturi meter
$2 * 10^{5} \leq \operatorname{Re}_{d 1} \leq 2 * 10^{6}$
$0.4 \leq \beta \leq 0.75$
for cast unit $\quad \mathrm{C}=0.984$
for machine units $\mathrm{C}=0.995$
Sonic nozzle $\quad \dot{m}_{\text {max }}=\rho_{1} A_{o} \sqrt{2 R T_{1}} \sqrt{\frac{k}{k+1}\left(\frac{2}{k+1}\right)^{2 /(k-1)}}$ where
k is the specific heat ratio
R is the gas constant in $\mathrm{J} / \mathrm{kg} . \mathrm{K}$
$\rho_{1}, T_{1}$ is the upstream density and temperature
Overall pressure losses: $\Delta \mathbf{p l o s s s}$, and power for the prime mover $\dot{W}=Q \frac{\Delta p_{\text {loss }}}{\eta}$
where $\eta$ is the prime mover efficiency
Laminar flow elements $Q=\frac{\pi d^{4}}{128 \mu} \frac{p_{1}-p_{2}}{L}$

## Vortex shedding

Strouhal number
$S t=f d / \bar{U}$

## Rotameter

$Q=A U=A\left[\frac{1}{C_{d}} \frac{2 g V_{b}}{A_{b}}\left(\frac{\rho_{b}}{\rho_{f}}-1\right)\right]^{1 / 2}$ where the flow area A is given by $A=\frac{\pi}{4}\left[(D+a y)^{2}-d^{2}\right]$
a=tube taper=Change of diameter over change of vertical distance y
Subscript brefers to the float. Subscript f refers to fluid
$\mathrm{A}_{\mathrm{b}}$ is the projected area of the float $=(\pi / 4) d^{2}$. D is the inlet diameter of the meter.


Figure 10.6 Flow coefficients for a square-edged orifice meter having flange pressure taps. (Compiled from data in [2]).


Figure 10.7 Expansion factors for common obstruction meters with $k=c_{p} / c_{v}=1.4$. (Courtesy of American Society of Mechanical Engineers, New York; compiled and reprinted from [2].)


Figure 10.11 Flow coefficients for an ASME long-radius nozzle with a throat pressure tap. (Compiled from [2].)

Table 10.1 Shedder Shape and Strouhal Number

${ }^{a}$ For Reynolds number $\operatorname{Re}_{d} \geq 10^{4}$. Strouhal number St $=f d / \bar{U}$.

## Ch. 11 Strain Measurements

Axial stress strain relation (Hook's law)
$\sigma_{a}=E_{m} \varepsilon_{a}$
Poission's ratio $\mathbf{v}_{\mathbf{p}}$

$v_{p}=\frac{\mid \text { lateral strain } \mid}{\mid \text { axial strain } \mid}=\frac{\varepsilon_{L}}{\varepsilon_{a}}$
Metallic gage
$R=\frac{\rho_{e} L}{A_{c}}, \quad \rho_{\mathrm{e}}=$ electric resistivity
$\frac{d R}{R}=\frac{d L}{L}\left(1+2 v_{p}\right)+\frac{d \rho_{e}}{\rho_{e}}$

$\frac{d R}{R}=\frac{d L}{L}\left(1+2 v_{p}+\pi_{1} E_{m}\right)$
where $\pi_{1}$ is called piezoresistance coefficient
$\pi_{1}=\frac{1}{E_{m}} \frac{d \rho_{e} / \rho_{e}}{d L / L}$
Gage factor GF is defined as
$G F=\frac{d R / R}{d L / L}=\frac{d R / R}{\varepsilon_{a}}$


Output voltage change $\mathrm{dE}_{0}$ due to bridge deflection
$\frac{\delta E_{o}}{E_{i}}=\frac{\delta R / R}{4+2(d R / R)} \approx \frac{\delta R / R}{4}=\frac{G F \varepsilon_{a}}{4}$

## Strains and stresses in plan area

$$
\begin{array}{ll}
\varepsilon_{y}=\frac{\sigma_{y}}{E_{m}}-v_{p} \frac{\sigma_{x}}{E_{m}} & \varepsilon_{x}=\frac{\sigma_{x}}{E_{m}}-v_{p} \frac{\sigma_{y}}{E_{m}} \\
\sigma_{x}=\frac{E_{m}\left(\varepsilon_{x}+v_{p} \varepsilon_{y}\right)}{1-v_{p}^{2}} & \sigma_{y}=\frac{E_{m}\left(\varepsilon_{y}+v_{p} \varepsilon_{x}\right)}{1-v_{p}^{2}}
\end{array}
$$

For thin walled vessels ( $\mathbf{t} / \mathbf{r}$ ) thickness/radius $<\mathbf{1 0}$
The relation between the pressure inside the vessel and the stresses is given by
$\sigma_{x}=\frac{P r}{t} \quad \sigma_{y}=\frac{P r}{2 t}$
$\sigma_{x}=2 \sigma_{y}$
$\varepsilon_{x}=\frac{\sigma_{x}}{E_{m}}\left(1-0.5 v_{p}\right)$
P is the pressure inside the vessel
$r$ is the radius of the vessel

t is the vessel's wall thickness

## Four arms of Wheatstone bridge

$\frac{\delta E_{o}}{E_{i}}=\frac{G F}{4}\left(\varepsilon_{1}-\varepsilon_{2}+\varepsilon_{4}-\varepsilon_{3}\right)$
Bridge constant $\kappa$
$\kappa=$ (Actual bridge output/Output of a single gauge on the bridge)
$\frac{\delta E_{o}}{E_{i}}=\frac{\kappa G F \varepsilon}{4}$


Table 11.1 Common Gauge Mountings


| Arrangement | Compensation <br> Provided | Bridge <br> Constant $\mathrm{\kappa}$ |
| :---: | :---: | :---: | :---: | :---: |

## Rosettes

A) $\mathbf{0 , 4 5 , 9 0}{ }^{\circ}$ Rosette

$$
\begin{aligned}
& \sigma_{\max }=\frac{E_{m}}{2}\left[\frac{\varepsilon_{1}+\varepsilon_{3}}{1-v_{p}}+\frac{1}{1+v_{p}} \sqrt{\left(\varepsilon_{1}-\varepsilon_{3}\right)^{2}+\left[2 \varepsilon_{2}-\left(\varepsilon_{1}+\varepsilon_{3}\right)\right]^{2}}\right] \\
& \sigma_{\min }= \frac{E_{m}}{2}\left[\frac{\varepsilon_{1}+\varepsilon_{3}}{1-v_{p}}-\frac{1}{1+v_{p}} \sqrt{\left(\varepsilon_{1}-\varepsilon_{3}\right)^{2}+\left[2 \varepsilon_{2}-\left(\varepsilon_{1}+\varepsilon_{3}\right)\right]^{2}}\right] \\
& \tau_{\max }=\frac{E_{m}}{2\left(1+v_{p}\right)} \sqrt{\left(\varepsilon_{1}-\varepsilon_{3}\right)^{2}+\left[2 \varepsilon_{2}-\left(\varepsilon_{1}+\varepsilon_{3}\right)\right]^{2}}
\end{aligned}
$$



The angle between the $x$-axis and the maximum principal stress is given by

$$
\phi=\frac{1}{2} \tan ^{-1} \frac{2 \varepsilon_{2}-\left(\varepsilon_{1}+\varepsilon_{3}\right)}{\varepsilon_{1}-\varepsilon_{3}}
$$

$\varphi$ in the first quadrant if $\varepsilon_{2}>\frac{\varepsilon_{1}+\varepsilon_{3}}{2}$, otherwise it is in the second quadrant
B) $\mathbf{0}, \mathbf{6 0}, 120^{\circ}$ Rosette

$$
\begin{aligned}
\sigma_{\max }, \sigma_{\min }= & \frac{E_{m}\left(\varepsilon_{1}+\varepsilon_{2}+\varepsilon_{3}\right)}{3(1-v)} \\
& \pm \frac{\sqrt{2} E_{m}}{3(1+v)}\left[\left(\varepsilon_{1}-\varepsilon_{2}\right)^{2}+\left(\varepsilon_{2}-\varepsilon_{3}\right)^{2}\right. \\
& \left.+\left(\varepsilon_{3}-\varepsilon_{1}\right)^{2}\right]^{\frac{1}{2}}
\end{aligned}
$$



$$
\begin{gathered}
\tau_{\max }=\frac{\sqrt{2} E_{m}}{3(1+v)}\left[\left(\varepsilon_{1}-\varepsilon_{2}\right)^{2}+\left(\varepsilon_{2}-\varepsilon_{3}\right)^{2}+\left(\varepsilon_{3}-\varepsilon_{1}\right)^{2}\right]^{\frac{1}{2}} \\
\varepsilon_{\max }, \varepsilon_{\min }=\frac{\varepsilon_{1}+\varepsilon_{2}+\varepsilon_{3}}{3} \pm \frac{\sqrt{2}}{3}\left[\left(\varepsilon_{1}-\varepsilon_{2}\right)^{2}+\left(\varepsilon_{2}-\varepsilon_{3}\right)^{2}+\left(\varepsilon_{3}-\varepsilon_{1}\right)^{2}\right]^{\frac{1}{2}} \\
\tan (2 \varphi)=\frac{\sqrt{3}\left(\varepsilon_{3}-\varepsilon_{2}\right)}{2 \varepsilon_{1}-\varepsilon_{2}-\varepsilon_{3}}
\end{gathered}
$$

$\varphi$ is in the first quadrant if $\varepsilon_{3}>\varepsilon_{2}$ otherwise it is in the second quadrant

## Ch. 7 Data Acquisition System

Resolution $=\frac{V_{\text {max }}-V_{\text {min }}}{2^{M}}$
Signal Noise ratio (SNR)
$\operatorname{SNR}(d B)=20 \log \left(2^{M}\right)$
Sampling rate
Nyquist Theorem
$f_{s} \geq 2 f_{x}$
$f_{x}=$ signal frequency, $f_{s}=$ sampling frequency

TABLE 17 Type T Thermocouple - thermoelectric voltage as a function of temperature $\left({ }^{\circ} \mathrm{C}\right)$; reference junctions at $0{ }^{\circ} \mathrm{C}$

| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | C |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Thermoelectric Voltage in Millivolts |  |  |  |  |  |  |  |  |  |  |  |  |
| -270 | -6.258 |  |  |  |  |  |  |  |  |  |  | -270 |
| -260 | -6.232 | -6.236 | -6.239 | -6.242 | -6.245 | -6.248 | -6.251 | -6.253 | -6.255 | -6.256 | -6.258 | -260 |
| -250 | -6.180 | -6.187 | -6.193 | -6.198 | -6.204 | -6.209 | -6.214 | -6.219 | -6.223 | -6.228 | -6.232 | -250 |
| -240 | -6.105 | -6.114 | -6.122 | -6.130 | -6.138 | -6.146 | -6.153 | -6.160 | -6.167 | -6.174 | -6.180 | -240 |
| -230 | -6.007 | -6.017 | -6.028 | -6.038 | -6.049 | -6.059 | -6.068 | -6.078 | -6.087 | -6.096 | -6.105 | -230 |
| -220 | -5.888 | -5.901 | -5.914 | -5.926 | -5.938 | -5.950 | -5.962 | -5.973 | -5.985 | -5.996 | -6.007 | -220 |
| -210 | -5.753 | -5.767 | -5.782 | -5.795 | -5.809 | -5.823 | -5.836 | -5.850 | -5.863 | -5.876 | -5.888 | -210 |
| -200 | -5.603 | -5.619 | -5.634 | -5.650 | -5.665 | -5.680 | -5.695 | -5.710 | -5.724 | -5.739 | -5.753 | -200 |
| -190 | -5.439 | -5.456 | -5.473 | -5.489 | -5.506 | -5.523 | -5.539 | -5.555 | -5.571 | -5.587 | -5.603 | -190 |
| -180 | -5.261 | -5.279 | -5.297 | -5.316 | -5.334 | -5.351 | -5.369 | -5.387 | -5.404 | -5.421 | -5.439 | -180 |
| -170 | -5.070 | -5.089 | -5.109 | -5.128 | -5.148 | -5.167 | -5.186 | -5.205 | -5.224 | -5.242 | -5.261 | -170 |
| -160 | -4.865 | -4.886 | -4.907 | -4.928 | -4.949 | -4.969 | -4.989 | -5.010 | -5.030 | -5.050 | -5.070 | -160 |
| -150 | -4.648 | -4.671 | -4.693 | -4.715 | -4.737 | -4.759 | -4.780 | -4.802 | -4.823 | -4.844 | -4.865 | -150 |
| -140 | -4.419 | -4.443 | -4.466 | -4.489 | -4.512 | -4.535 | -4.558 | -4.581 | -4.604 | -4.626 | -4.648 | -140 |
| -130 | -4.177 | -4.202 | -4.226 | -4.251 | -4.275 | -4.300 | -4.324 | -4.348 | -4.372 | -4.395 | -4.419 | -130 |
| -120 | -3.923 | -3.949 | -3.975 | -4.000 | -4.026 | -4.052 | -4.077 | -4.102 | -4.127 | -4.152 | -4.177 | -120 |
| -110 | -3.657 | -3.684 | -3.711 | -3.738 | -3.765 | -3.791 | -3.818 | -3.844 | -3.871 | -3.897 | -3.923 | -110 |
| -100 | -3.379 | -3.407 | -3.435 | -3.463 | -3.491 | -3.519 | -3.547 | -3.574 | -3.602 | -3.629 | -3.657 | -100 |
| -90 | -3.089 | -3.118 | -3.148 | -3.177 | -3.206 | -3.235 | -3.264 | -3.293 | -3.322 | -3.350 | -3.379 | -90 |
| -80 | -2.788 | -2.818 | -2.849 | -2.879 | -2.910 | -2.940 | -2.970 | -3.000 | -3.030 | -3.059 | -3.089 | -80 |
| -70 | -2.476 | -2.507 | -2.539 | -2.571 | -2.602 | -2.633 | -2.664 | -2.695 | -2.726 | -2.757 | -2.788 | -70 |
| -60 | -2.153 | -2.186 | -2.218 | -2.251 | -2.283 | -2.316 | -2.348 | -2.380 | -2.412 | -2.444 | -2.476 | -60 |
| -50 | -1.819 | -1.853 | -1.887 | -1.920 | -1.954 | -1.987 | -2.021 | -2.054 | -2.087 | -2.120 | -2.153 | -50 |
| -40 | -1.475 | -1.510 | -1.545 | -1.579 | -1.614 | -1.648 | -1.683 | -1.717 | -1.751 | -1.785 | -1.819 | -40 |
| -30 | -1.121 | -1.157 | -1.192 | -1.228 | -1.264 | -1.299 | -1.335 | -1.370 | -1.405 | -1.440 | -1.475 | -30 |
| -20 | -0.757 | -0.794 | -0.830 | -0.867 | -0.904 | -0.940 | -0.976 | -1.013 | -1.049 | -1.085 | -1.121 | -20 |
| -10 | -0.383 | -0.421 | -0.459 | -0.496 | -0.534 | -0.571 | -0.608 | -0.646 | -0.683 | -0.720 | -0.757 | -10 |
| 0 | 0.000 | -0.039 | -0.077 | -0.116 | -0.154 | -0.193 | -0.231 | -0.269 | -0.307 | -0.345 | -0.383 | 0 |
| 0 | 0.000 | 0.039 | 0.078 | 0.117 | 0.156 | 0.195 | 0.234 | 0.273 | 0.312 | 0.352 | 0.391 | 0 |
| 10 | 0.391 | 0.431 | 0.470 | 0.510 | 0.549 | 0.589 | 0.629 | 0.669 | 0.709 | 0.749 | 0.790 | 10 |
| 20 | 0.790 | 0.830 | 0.870 | 0.911 | 0.951 | 0.992 | 1.033 | 1.074 | 1.114 | 1.155 | 1.196 | 20 |
| 30 | 1.196 | 1.238 | 1.279 | 1.320 | 1.362 | 1.403 | 1.445 | 1.486 | 1.528 | 1.570 | 1.612 | 30 |
| 40 | 1.612 | 1.654 | 1.696 | 1.738 | 1.780 | 1.823 | 1.865 | 1.908 | 1.950 | 1.993 | 2.036 | 40 |
| 50 | 2.036 | 2.079 | 2.122 | 2.165 | 2.208 | 2.251 | 2.294 | 2.338 | 2.381 | 2.425 | 2.468 | 50 |
| 60 | 2.468 | 2.512 | 2.556 | 2.600 | 2.643 | 2.687 | 2.732 | 2.776 | 2.820 | 2.864 | 2.909 | 60 |
| 70 | 2.909 | 2.953 | 2.998 | 3.043 | 3.087 | 3.132 | 3.177 | 3.222 | 3.267 | 3.312 | 3.358 | 70 |
| 80 | 3.358 | 3.403 | 3.448 | 3.494 | 3.539 | 3.585 | 3.631 | 3.677 | 3.722 | 3.768 | 3.814 | 80 |
| 90 | 3.814 | 3.860 | 3.907 | 3.953 | 3.999 | 4.046 | 4.092 | 4.138 | 4.185 | 4.232 | 4.279 | 90 |
| 100 | 4.279 | 4.325 | 4.372 | 4.419 | 4.466 | 4.513 | 4.561 | 4.608 | 4.655 | 4.702 | 4.750 | 100 |
| 110 | 4.750 | 4.798 | 4.845 | 4.893 | 4.941 | 4.988 | 5.036 | 5.084 | 5.132 | 5.180 | 5.228 | 110 |
| 120 | 5.228 | 5.277 | 5.325 | 5.373 | 5.422 | 5.470 | 5.519 | 5.567 | 5.616 | 5.665 | 5.714 | 120 |
| 130 | 5.714 | 5.763 | 5.812 | 5.861 | 5.910 | 5.959 | 6.008 | 6.057 | 6.107 | 6.156 | 6.206 | 130 |
| 140 | 6.206 | 6.255 | 6.305 | 6.355 | 6.404 | 6.454 | 6.504 | 6.554 | 6.604 | 6.654 | 6.704 | 140 |
| 150 | 6.704 | 6.754 | 6.805 | 6.855 | 6.905 | 6.956 | 7.006 | 7.057 | 7.107 | 7.158 | 7.209 | 150 |
| 160 | 7.209 | 7.260 | 7.310 | 7.361 | 7.412 | 7.463 | 7.515 | 7.566 | 7.617 | 7.668 | 7.720 | 160 |
| 170 | 7.720 | 7.771 | 7.823 | 7.874 | 7.926 | 7.977 | 8.029 | 8.081 | 8.133 | 8.185 | 8.237 | 170 |
| 180 | 8.237 | 8.289 | 8.341 | 8.393 | 8.445 | 8.497 | 8.550 | 8.602 | 8.654 | 8.707 | 8.759 | 180 |
| 190 | 8.759 | 8.812 | 8.865 | 8.917 | 8.970 | 9.023 | 9.076 | 9.129 | 9.182 | 9.235 | 9.288 | 190 |


| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | ${ }^{\circ} \mathrm{C}$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |

TABLE 7 Type J Thermocouple - thermoelectric voltage as a function of

| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | ${ }^{\circ} \mathrm{C}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Thermoelectric Voltage in Millivolts |  |  |  |  |  |  |  |  |  |  |  |  |
| -210 | -8.095 |  |  |  |  |  |  |  |  |  |  | -210 |
| -200 | -7.890 | -7.912 | -7.934 | $-7.955$ | -7.976 | -7.996 | -8.017 | -8.037 | -8.057 | -8.076 | -8.095 | -200 |
| -190 | -7.659 | -7.683 | $-7.707$ | $-7.731$ | $-7.755$ | $-7.778$ | $-7.801$ | -7.824 | -7.846 | $-7.868$ | $-7.890$ | -190 |
| -180 | -7.403 | -7.429 | -7.456 | -7.482 | -7.508 | -7.534 | -7.559 | -7.585 | -7.610 | -7.634 | -7.659 | -180 |
| -170 | -7.123 | -7.152 | -7.181 | -7.209 | -7.237 | -7.265 | -7.293 | -7.321 | -7.348 | -7.376 | -7.403 | -170 |
| -160 | -6.821 | -6.853 | -6.883 | -6.914 | -6.944 | -6.975 | -7.005 | -7.035 | -7.064 | -7.094 | -7.123 | -160 |
| -150 | -6.500 | -6.533 | -6.566 | -6.598 | -6.631 | -6.663 | -6.695 | -6.727 | -6.759 | -6.790 | -6.821 | -150 |
| -140 | -6.159 | -6.194 | -6.229 | -6.263 | -6.298 | -6.332 | -6.366 | -6.400 | -6.433 | -6.467 | -6.500 | -140 |
| -130 | -5.801 | -5.838 | -5.874 | -5.910 | -5.946 | -5.982 | -6.018 | -6.054 | -6.089 | -6.124 | -6.159 | -130 |
| -120 | -5.426 | -5.465 | -5.503 | -5.541 | -5.578 | -5.616 | -5.653 | -5.690 | -5.727 | -5.764 | -5.801 | -120 |
| -110 | -5.037 | -5.076 | -5.116 | $-5.155$ | -5.194 | $-5.233$ | -5.272 | -5.311 | -5.350 | -5.388 | -5.426 | -110 |
| -100 | -4.633 | -4.674 | -4.714 | -4.755 | -4.796 | $-4.836$ | $-4.877$ | -4.917 | -4.957 | $-4.997$ | -5.037 | -100 |
| -90 | -4.215 | -4.257 | -4.300 | -4.342 | -4.384 | -4.425 | -4.467 | -4.509 | -4.550 | -4.591 | -4.633 | -90 |
| -80 | -3.786 | -3.829 | -3.872 | -3.916 | -3.959 | -4.002 | -4.045 | -4.088 | -4.130 | -4.173 | -4.215 | -80 |
| -70 | -3.344 | -3.389 | -3.434 | -3.478 | -3.522 | -3.566 | -3.610 | -3.654 | -3.698 | -3.742 | -3.786 | -70 |
| -60 | -2.893 | -2.938 | -2.984 | -3.029 | -3.075 | -3.120 | -3.165 | -3.210 | -3.255 | -3.300 | -3.344 | -60 |
| -50 | $-2.431$ | $-2.478$ | -2.524 | $-2.571$ | $-2.617$ | $-2.663$ | -2.709 | -2.755 | -2.801 | $-2.847$ | $-2.893$ | -50 |
| -40 | -1.961 | -2.008 | $-2.055$ | -2.103 | $-2.150$ | $-2.197$ | -2.244 | $-2.291$ | -2.338 | -2.385 | -2.431 | -40 |
| -30 | -1.482 | -1.530 | -1.578 | -1.626 | -1.674 | -1.722 | -1.770 | -1.818 | -1.865 | -1.913 | -1.961 | -30 |
| -20 | -0.995 | -1.044 | -1.093 | -1.142 | -1.190 | -1.239 | -1.288 | -1.336 | -1.385 | -1.433 | -1.482 | -20 |
| -10 | -0.501 | -0.550 | -0.600 | -0.650 | -0.699 | -0.749 | -0.798 | -0.847 | -0.896 | -0.946 | -0.995 | -10 |
| 0 | 0.000 | -0.050 | $-0.101$ | $-0.151$ | -0.201 | $-0.251$ | -0.301 | -0.351 | -0.401 | -0.451 | -0.501 | 0 |
| 0 | 0.000 | 0.050 | 0.101 | 0.151 | 0.202 | 0.253 | 0.303 | 0.354 | 0.405 | 0.456 | 0.507 | 0 |
| 10 | 0.507 | 0.558 | 0.609 | 0.660 | 0.711 | 0.762 | 0.814 | 0.865 | 0.916 | 0.968 | 1.019 | 10 |
| 20 | 1.019 | 1.071 | 1.122 | 1.174 | 1.226 | 1.277 | 1.329 | 1.381 | 1.433 | 1.485 | 1.537 | 20 |
| 30 | 1.537 | 1.589 | 1.641 | 1.693 | 1.745 | 1.797 | 1.849 | 1.902 | 1.954 | 2.006 | 2.059 | 30 |
| 40 | 2.059 | 2.111 | 2.164 | 2.216 | 2.269 | 2.322 | 2.374 | 2.427 | 2.480 | 2.532 | 2.585 | 40 |
| 50 | 2.585 | 2.638 | 2.691 | 2.744 | 2.797 | 2.850 | 2.903 | 2.956 | 3.009 | 3.062 | 3.116 | 50 |
| 60 | 3.116 | 3.169 | 3.222 | 3.275 | 3.329 | 3.382 | 3.436 | 3.489 | 3.543 | 3.596 | 3.650 | 60 |
| 70 | 3.650 | 3.703 | 3.757 | 3.810 | 3.864 | 3.918 | 3.971 | 4.025 | 4.079 | 4.133 | 4.187 | 70 |
| 80 | 4.187 | 4.240 | 4.294 | 4.348 | 4.402 | 4.456 | 4.510 | 4.564 | 4.618 | 4.672 | 4.726 | 80 |
| 90 | 4.726 | 4.781 | 4.835 | 4.889 | 4.943 | 4.997 | 5.052 | 5.106 | 5.160 | 5.215 | 5.269 | 90 |
| 100 | 5.269 | 5.323 | 5.378 | 5.432 | 5.487 | 5.541 | 5.595 | 5.650 | 5.705 | 5.759 | 5.814 | 100 |
| 110 | 5.814 | 5.868 | 5.923 | 5.977 | 6.032 | 6.087 | 6.141 | 6.196 | 6.251 | 6.306 | 6.360 | 110 |
| 120 | 6.360 | 6.415 | 6.470 | 6.525 | 6.579 | 6.634 | 6.689 | 6.744 | 6.799 | 6.854 | 6.909 | 120 |
| 130 | 6.909 | 6.964 | 7.019 | 7.074 | 7.129 | 7.184 | 7.239 | 7.294 | 7.349 | 7.404 | 7.459 | 130 |
| 140 | 7.459 | 7.514 | 7.569 | 7.624 | 7.679 | 7.734 | 7.789 | 7.844 | 7.900 | 7.955 | 8.010 | 140 |
| 150 | 8.010 | 8.065 | 8.120 | 8.175 | 8.231 | 8.286 | 8.341 | 8.396 | 8.452 | 8.507 | 8.562 | 150 |
| 160 | 8.562 | 8.618 | 8.673 | 8.728 | 8.783 | 8.839 | 8.894 | 8.949 | 9.005 | 9.060 | 9.115 | 160 |
| 170 | 9.115 | 9.171 | 9.226 | 9.282 | 9.337 | 9.392 | 9.448 | 9.503 | 9.559 | 9.614 | 9.669 | 170 |
| 180 | 9.669 | 9.725 | 9.780 | 9.836 | 9.891 | 9.947 | 10.002 | 10.057 | 10.113 | 10.168 | 10.224 | 180 |
| 190 | 10.224 | 10.279 | 10.335 | 10.390 | 10.446 | 10.501 | 10.557 | 10.612 | 10.668 | 10.723 | 10.779 | 190 |
| 200 | 10.779 | 10.834 | 10.890 | 10.945 | 11.001 | 11.056 | 11.112 | 11.167 | 11.223 | 11.278 | 11.334 | 200 |
| 210 | 11.334 | 11.389 | 11.445 | 11.501 | 11.556 | 11.612 | 11.667 | 11.723 | 11.778 | 11.834 | 11.889 | 210 |
| 220 | 11.889 | 11.945 | 12.000 | 12.056 | 12.111 | 12.167 | 12.222 | 12.278 | 12.334 | 12.389 | 12.445 | 220 |
| 230 | 12.445 | 12.500 | 12.556 | 12.611 | 12.667 | 12.722 | 12.778 | 12.833 | 12.889 | 12.944 | 13.000 | 230 |
| 240 | 13.000 | 13.056 | 13.111 | 13.167 | 13.222 | 13.278 | 13.333 | 13.389 | 13.444 | 13.500 | 13.555 | 240 |


| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 6 | 6 | 7 | 8 | 9 | 10 | ${ }^{\circ} \mathrm{C}$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |

TABLE 9 Type K Thermocouple - thermoelectric voitage as a function of tomperature ( ${ }^{\circ}$ ); reference junctions at $0^{\circ} \mathrm{C}$

| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | C |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Thermoelectric Voltage in Millivolts |  |  |  |  |  |  |  |  |  |  |  |  |
| -270 | -6.458 |  |  |  |  |  |  |  |  |  |  | -270 |
| -260 | -6.411 | -6.444 | -6.446 | -6.448 | -6.450 | -6.452 | -6.453 | -6.455 | -6.456 | -6.457 | -6.458 | -260 |
| -250 | -6.404 | -6.408 | -6.413 | -6.417 | -6.421 | -6.425 | -6.429 | -6.432 | -6.435 | -6.438 | -6.441 | -250 |
| $-240$ | -6.344 | -6.351 | -6.358 | -6.364 | -6.370 | $-6.377$ | -6.382 | -6.388 | -6.393 | -6.399 | -6.404 | -240 |
| -230 | -6.262 | -6.271 | -6.280 | -6.289 | -6.297 | -6.306 | -6.314 | -6.322 | -6.329 | -6.337 | -6.344 | -230 |
| -220 | -6.158 | -6.170 | -6.181 | -6.192 | -6.202 | -6.213 | -6.223 | -6.233 | -6.243 | -6.252 | -6.262 | -220 |
| -210 | -6.035 | -6.048 | -6.061 | -6.074 | -6.087 | -6.099 | -6.111 | -6.123 | -6.135 | -6.147 | -6.158 | -210 |
| -200 | -5.891 | -5.907 | $-5.922$ | -5.936 | -5.951 | $-5.965$ | $-5.980$ | -5.994 | $-6.007$ | -6.021 | -6.035 | -200 |
| -190 | -5.730 | -5.747 | -5.763 | -5.780 | -5.797 | -5.813 | -5.829 | -5.845 | -5.861 | -5.876 | -5.891 | -190 |
| -180 | -5.550 | -5.569 | -5.588 | -5.606 | $-5.624$ | $-5.642$ | $-5.660$ | -5.678 | -5.695 | -5.713 | -5.730 | -180 |
| -170 | -5.354 | -5.374 | -5.395 | -5.415 | -5.435 | -5.454 | -5.474 | -5.493 | -5.512 | -5.531 | -5.550 | -170 |
| -160 | -5.141 | -5.163 | -5.185 | -5.207 | -5.228 | $-5.250$ | -5.271 | -5.292 | -5.313 | -5.333 | -5.354 | -160 |
| -150 | -4.913 | -4.936 | -4.960 | -4.983 | $-5.006$ | -5.029 | $-5.052$ | -5.074 | $-5.097$ | -5.119 | -5.141 | -150 |
| -140 | -4.669 | -4.894 | -4.719 | -4.744 | -4.768 | -4.793 | -4.817 | -4.841 | -4.865 | -4.889 | -4.913 | -140 |
| -130 | -4.411 | -4.437 | -4.463 | -4.490 | -4.516 | -4.542 | -4.567 | -4.593 | -4.618 | -4.644 | -4.669 | -130 |
| -120 | -4.138 | -4.166 | -4.194 | -4.221 | -4.249 | -4.276 | -4.303 | -4.330 | -4.357 | -4.384 | -4.411 | -120 |
| -110 | -3.852 | -3.882 | -3.911 | -3.939 | -3.968 | -3.997 | -4.025 | -4.054 | -4.082 | -4.110 | -4.138 | -110 |
| -100 | -3.554 | -3.584 | -3.614 | -3.645 | -3.675 | -3.705 | -3.734 | -3.764 | -3.794 | $-3.823$ | -3.852 | -100 |
| -90 | -3.243 | -3.274 | -3.306 | -3.337 | -3.368 | $-3.400$ | -3.431 | -3.462 | -3.492 | -3.523 | -3.554 | -90 |
| -80 | -2.920 | -2.953 | -2.986 | -3.018 | -3.050 | -3.083 | -3.115 | -3.147 | -3.179 | -3.211 | -3.243 | -80 |
| -70 | -2.587 | -2.820 | -2.654 | -2.688 | -2.721 | -2.755 | -2.788 | -2.821 | -2.854 | -2.887 | -2.920 | -70 |
| -60 | -2.243 | -2.278 | -2.312 | -2.347 | -2.382 | -2.416 | -2.450 | -2.485 | -2.519 | -2.553 | -2.587 | -60 |
| -50 | -1.889 | -1.925 | -1.961 | -1.996 | -2.032 | -2.067 | -2.103 | -2.138 | -2.173 | -2.208 | -2.243 | -50 |
| -40 | -1.527 | -1.564 | -1.600 | -1.637 | -1.673 | -1.709 | -1.745 | -1.782 | -1.818 | -1.854 | -1.899 | -40 |
| -30 | -1.156 | -1.194 | -1.231 | -1.268 | -1.305 | -1.343 | -1.380 | -1.417 | -1.453 | -1.490 | -1.527 | -30 |
| -20 | -0.778 | -0.816 | -0.854 | -0.892 | -0.930 | -0.968 | -1.006 | -1.043 | -1.081 | -1.119 | -1.156 | -20 |
| -10 | -0.392 | -0.431 | -0.470 | -0.508 | -0.547 | -0.586 | -0.624 | -0.663 | -0.701 | -0.739 | -0.778 | -10 |
| 0 | 0.000 | -0.039 | -0.079 | -0.118 | $-0.157$ | $-0.197$ | -0.236 | -0.275 | -0.314 | $-0.353$ | -0.392 | 0 |
| 0 | 0.000 | 0.039 | 0.079 | 0.119 | 0.158 | 0.198 | 0.238 | 0.277 | 0.317 | 0.357 | 0.397 | 0 |
| 10 | 0.397 | 0.437 | 0.477 | 0.517 | 0.557 | 0.597 | 0.637 | 0.677 | 0.718 | 0.758 | 0.798 | 10 |
| 20 | 0.798 | 0.838 | 0.879 | 0.919 | 0.960 | 1.000 | 1.041 | 1.081 | 1.122 | 1.163 | 1.203 | 20 |
| 30 | 1.203 | 1.244 | 1.285 | 1.326 | 1.366 | 1.407 | 1.448 | 1.489 | 1.530 | 1.571 | 1.612 | 30 |
| 40 | 1.612 | 1.653 | 1.694 | 1.735 | 1.776 | 1.817 | 1.858 | 1.899 | 1.941 | 1.982 | 2.023 | 40 |
| 50 | 2.023 | 2.064 | 2.106 | 2.147 | 2.188 | 2.230 | 2.271 | 2.312 | 2.354 | 2.395 | 2.436 | 50 |
| 60 | 2.436 | 2.478 | 2.519 | 2.561 | 2.602 | 2.644 | 2.685 | 2.727 | 2.768 | 2.810 | 2.851 | 60 |
| 70 | 2.851 | 2.893 | 2.934 | 2.976 | 3.017 | 3.059 | 3.100 | 3.142 | 3.184 | 3.225 | 3.267 | 70 |
| 80 | 3.267 | 3.308 | 3.350 | 3.391 | 3.433 | 3.474 | 3.516 | 3.557 | 3.599 | 3.640 | 3.682 | 80 |
| 90 | 3.682 | 3.723 | 3.765 | 3.806 | 3.848 | 3.889 | 3.931 | 3.972 | 4.013 | 4.055 | 4.096 | 90 |
| 100 | 4.096 | 4.138 | 4.179 | 4.220 | 4.262 | 4.303 | 4.344 | 4.385 | 4.427 | 4.468 | 4.509 | 100 |
| 110 | 4.509 | 4.550 | 4.591 | 4.633 | 4.674 | 4.715 | 4.756 | 4.797 | 4.838 | 4.879 | 4.920 | 110 |
| 120 | 4.920 | 4.961 | 5.002 | 5.043 | 5.084 | 5.124 | 5.165 | 5.206 | 5.247 | 5.288 | 5.328 | 120 |
| 130 | 5.328 | 5.369 | 5.410 | 5.450 | 5.491 | 5.532 | 5.572 | 5.613 | 5.653 | 5.694 | 5.735 | 130 |
| 140 | 5.735 | 5.775 | 5.815 | 5.856 | 5.896 | 5.937 | 5.977 | 6.017 | 6.058 | 6.098 | 6.138 | 140 |
| 150 | 6.138 | 6.179 | 6.219 | 6.259 | 6.299 | 6.339 | 6.380 | 6.420 | 6.460 | 6.500 | 6.540 | 150 |
| 160 | 6.540 | 6.580 | 6.620 | 6.660 | 6.701 | 6.741 | 6.781 | 6.821 | 6.861 | 6.901 | 6.941 | 160 |
| 170 | 6.941 | 6.981 | 7.021 | 7.060 | 7.100 | 7.140 | 7.180 | 7.220 | 7.260 | 7.300 | 7.340 | 170 |
| 180 | 7.340 | 7.380 | 7.420 | 7.460 | 7.500 | 7.540 | 7.579 | 7.619 | 7.659 | 7.699 | 7.739 | 180 |
| 190 | 7.739 | 7.779 | 7.819 | 7.859 | 7.899 | 7.939 | 7.979 | 8.019 | 8.059 | 8.099 | 8.138 | 190 |


| ${ }^{\circ} \mathrm{C}$ | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | ${ }^{\circ} \mathrm{C}$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |

